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(54) **COMPOUND TRANSMISSION**

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See application file for complete search history.

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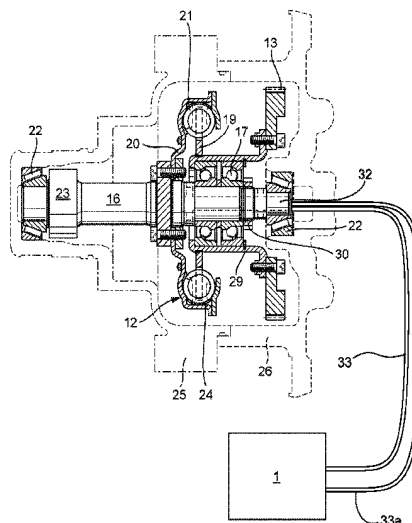
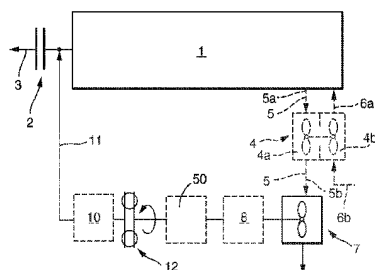
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(57) **ABSTRACT**

An internal combustion engine comprises includes a turbo-compound and the known Föttinger-coupling is replaced by a torsion vibration damper. The Föttinger-coupling, which is used to transmit power, has high losses in power when it is necessary to have a differential rotational speed between the input side and the output side, that is, the appearance of a slip. The losses are not used in a torsion vibration damper which has at least the same quality as a Föttinger-coupling.

17 Claims, 6 Drawing Sheets



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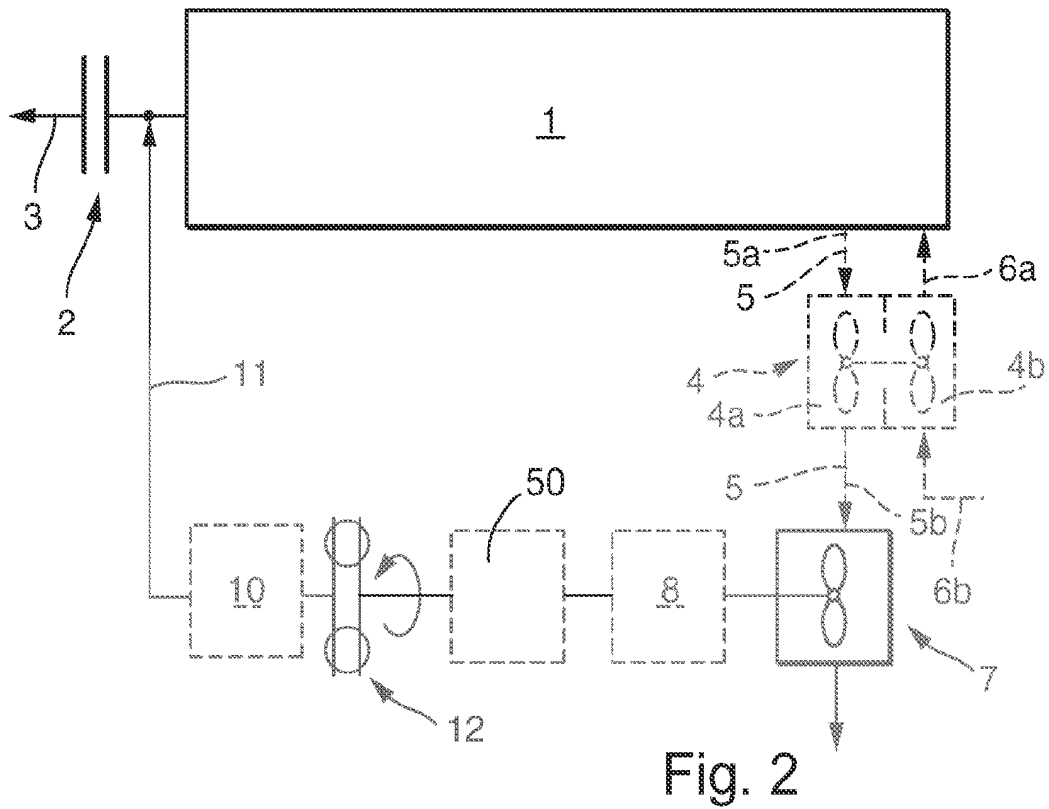
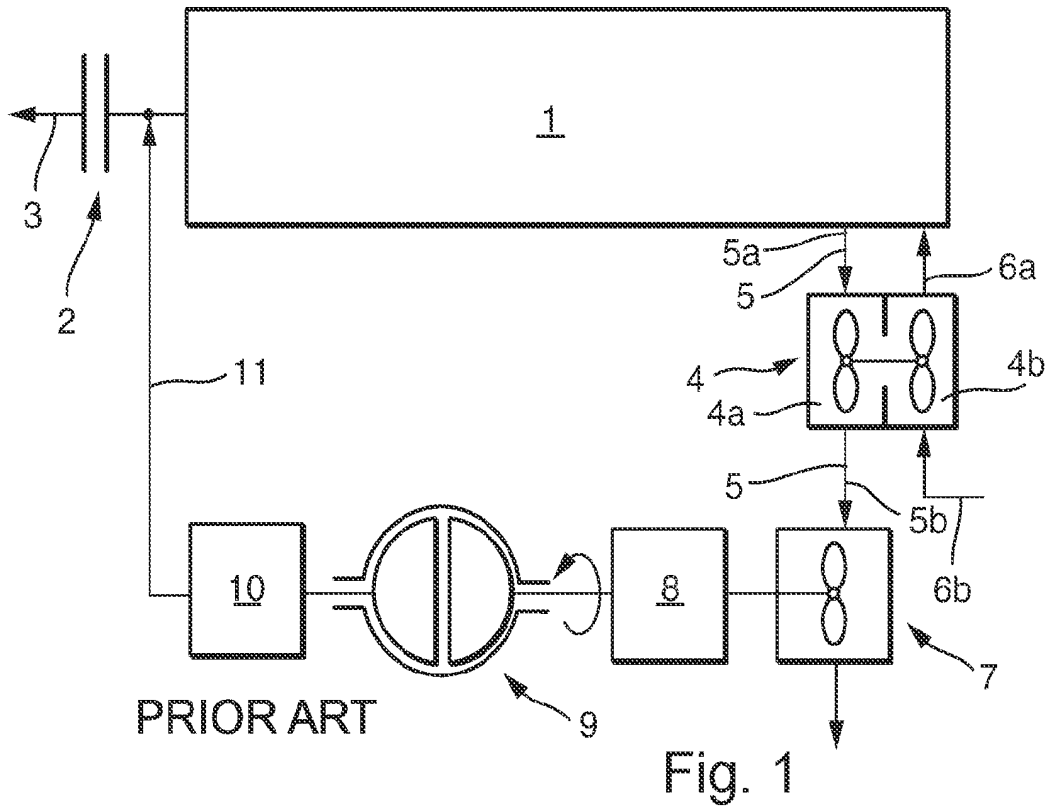
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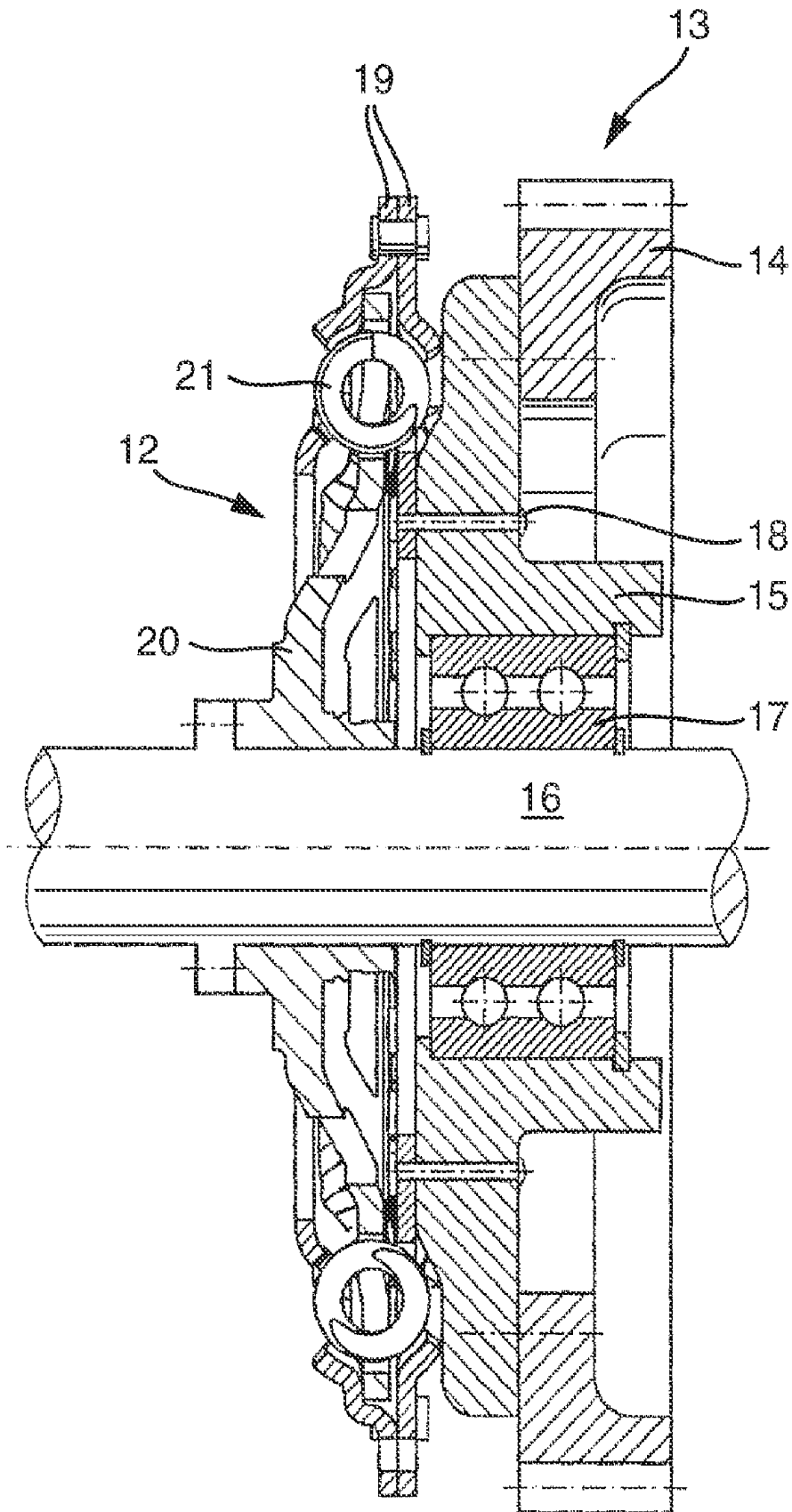


Fig. 3

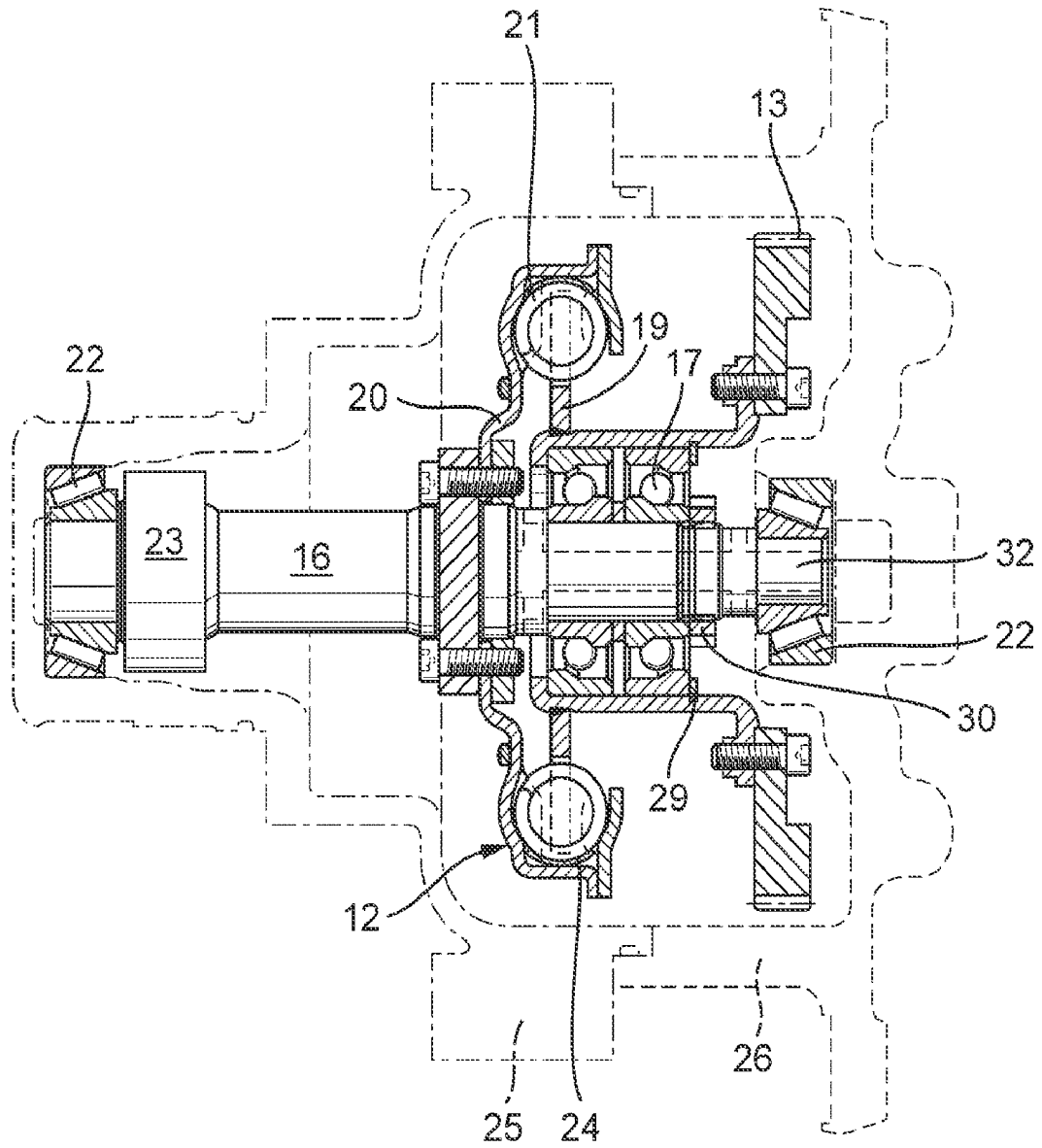


Fig. 4

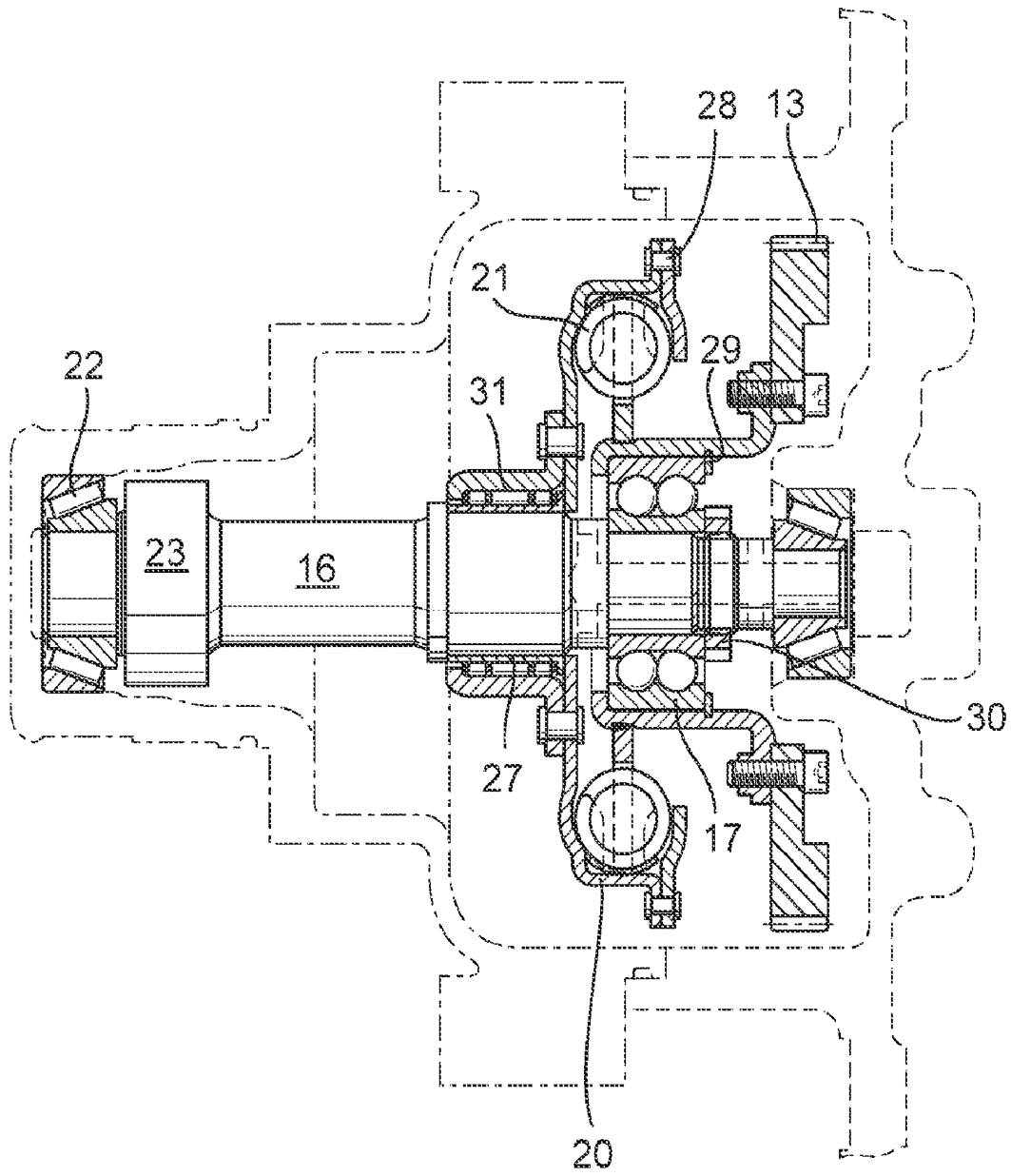


Fig. 5

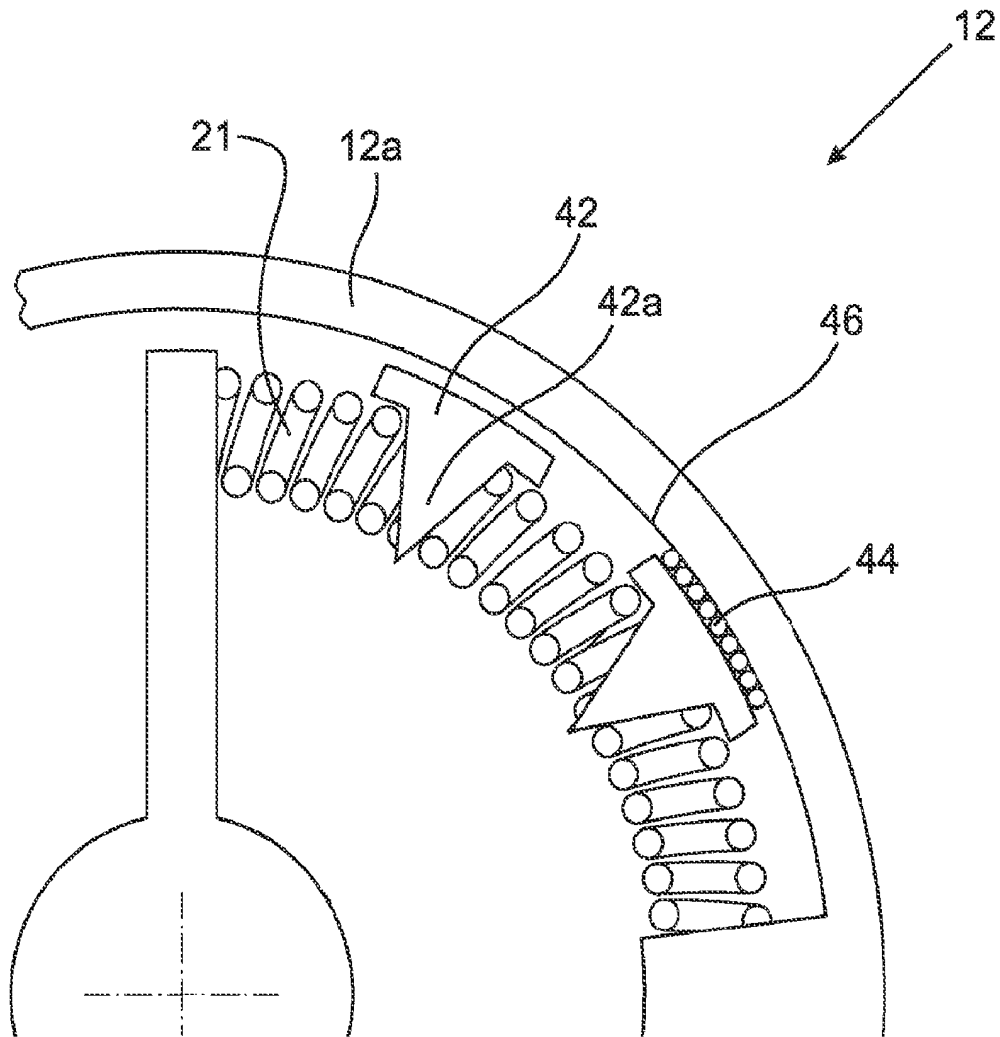


Fig. 6

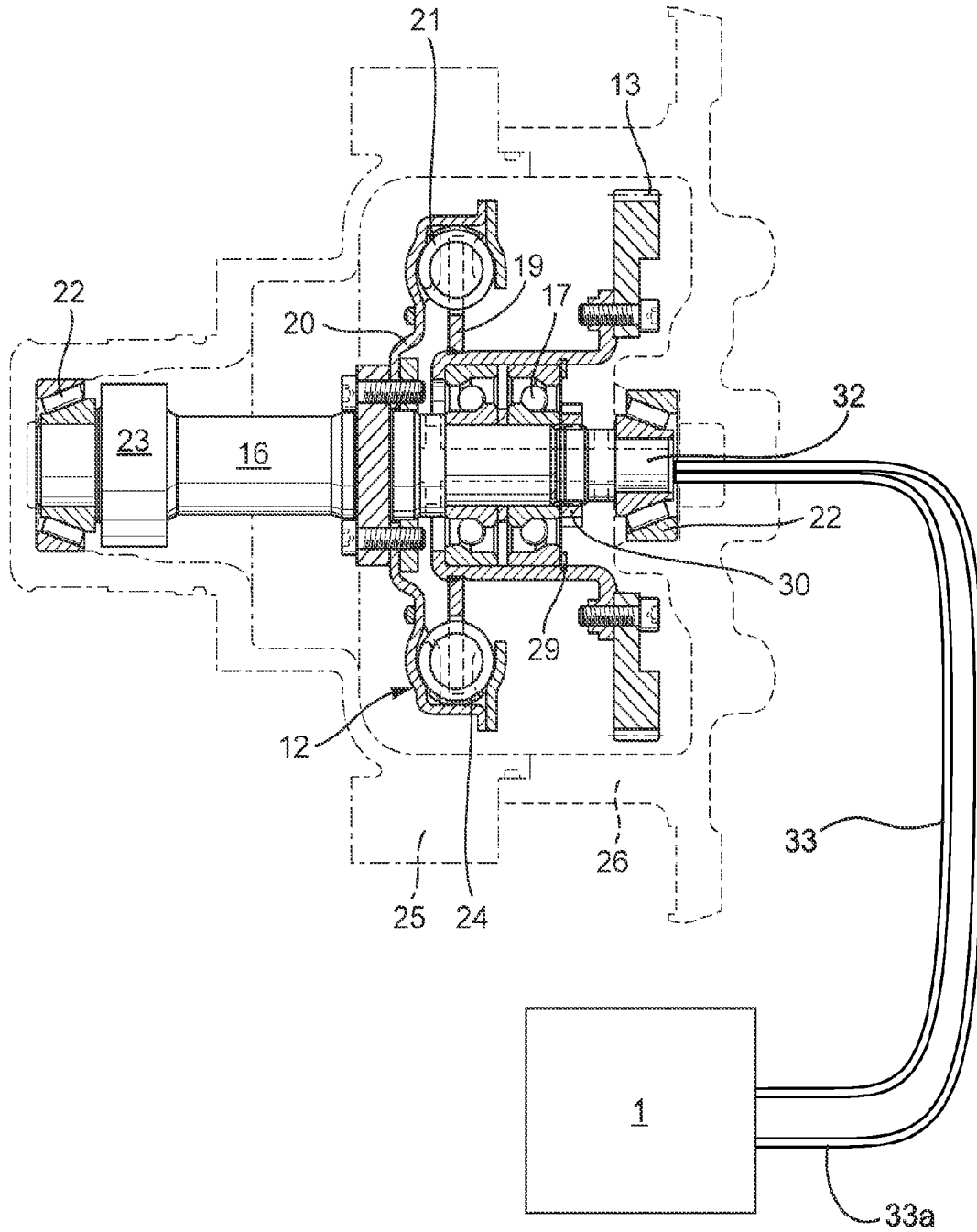


Fig. 7

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COMPOUND TRANSMISSIONCROSS REFERENCE TO RELATED
APPLICATIONS

This application claims the benefit of International Application PCT/DE2006/000519, filed 24 Mar. 2006, which claims priority from German Application DE 10 2005 014 000.9, filed 26 Mar. 2005, said applications are incorporated-by-reference herein.

FIELD OF THE INVENTION

The invention relates to a compound drive in combination with a combustion engine.

BACKGROUND OF THE INVENTION

Compound drives are also called turbo-compounds. In such compound drives, the stream of exhaust gas from a combustion engine—in many cases a diesel engine—is routed through the exhaust gas turbine of a turbocharger. The charge air is fed to the combustion engine through a blower that is connected to the exhaust gas turbine with a rotationally fixed connection. As the exhaust gases continue on their way, in the existing art—in combination with a compound drive—they pass through a second turbine. This second turbine may transmit its rotational energy to a reduction gear, which is then connected in turn to a Föttinger coupling. After the Föttinger coupling there can again be a reduction gear, in order to further reduce the speed of rotation. Finally, the rotational energy is led into the crankshaft or into the centrifugal mass of the combustion engine. Through such a design, the energy content of the exhaust gases is used to increase the drive energy of the combustion engine.

Non-uniformities of rotation between the crankshaft and the turbocharger are evened out through the use of the Föttinger coupling. Otherwise a rigid power train of the composite drive would convey non-uniformities of rotation of the crankshaft all the way to the power turbine, which would lead to significant torsion vibration problems.

The design of a Föttinger coupling is very complex, which also makes this solution very costly. Furthermore, the efficiency is not optimal, due to the slippage inherent in the system.

SUMMARY OF THE INVENTION

The object of the invention is therefore to provide a composite drive which both minimizes torsion vibrations and is economical.

The problem is solved by employing a torsion vibration damper instead of a Föttinger coupling in a compound drive. A torsion vibration damper may have various designs here. In a first design, the torsion vibration damper comprises an input part and an output part, there being extensively acting energy storage devices (for example in the form of springs) situated between the input and output parts.

In a second design of the torsion vibration damper there are also an input part and an output part present, but rolling elements move on imagined ramps in both directions between the input and output parts. The input and output parts here are braced axially against each other (for example by means of a diaphragm spring).

In a Föttinger coupling, the rotational coupling from the pump side to the turbine side takes place by means of the hydrodynamic principle. That also makes it possible to elimi-

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nate torsional vibrations—at least partially. Since because of the hydrodynamic principle no rotationally fixed connection exists within the Föttinger coupling, the rotational speeds can fluctuate between the pump side and the turbine side without the transmission of torque being disrupted.

In contrast to this, in a compound drive without Föttinger coupling but having a torsion vibration damper there is always a rotationally fixed connection. However, a torsion vibration damper permits only a relatively small relative angle of rotation between the input and output parts. These angle dimensions can be a maximum of $\pm 90^\circ$. In other words: With a torsion vibration damper, the rotary motions of the input and output parts—aside from the superimposed vibrations—are always rotationally synchronous. With a torsion vibration damper, “overtaking” of the output part by the input part is not possible. Given these facts, it is therefore all the more surprising that a compound drive with a torsion vibration damper instead of a Föttinger coupling can be realized.

The invention will now be explained in greater detail on the basis of the figures.

BRIEF DESCRIPTION OF THE DRAWINGS

The figures show the following:

FIG. 1: A schematic depiction of the existing art;

FIG. 2: a schematic depiction according to the invention;

FIG. 3: a cross sectional depiction of a torsion vibration damper;

FIG. 4: a cross section through another design of a torsion vibration damper;

FIG. 5: a cross section through a torsion vibration damper with freewheeling;

FIG. 6: a schematic view of the energy storage devices at the outermost diameter of the torsion vibration damper; and,

FIG. 7: a partial schematic view of the oil lubricant system of the compound drive of the present invention connected to the oil lubrication system of the internal combustion engine.

DESCRIPTION OF THE PREFERRED
EMBODIMENTS

In FIG. 1, a combustion engine 1 is connected to a turbocharger 4 through an exhaust gas line 5a. Turbocharger 4 is subdivided into the exhaust gas turbine 4a and the charge air turbine 4b. During operation of turbocharger 4, a charge air stream 6b passes through charge air turbine 4b and becomes charge air stream 6a, which is blown into combustion engine 1.

Farther along in exhaust gas line 5 is exhaust gas line 5b, which is flow-connected to a compound drive turbine 7. A second yield is obtained here from the exhaust air, the intent here being to obtain rotational energy for the compound drive. Compound drive turbine 7 is connected to a reduction gear by a rotationally fixed connection. The intent of this reduction gear 8 is to reduce the high speed of the compound drive turbine 7 to the nominal speed of the down-line Föttinger coupling 9. The right side of Föttinger coupling 9 here is the pump, while the left half shell of the Föttinger coupling embodies the turbine. Another reduction gear 10 is connected by a rotationally fixed connection to the left side of the Föttinger coupling—i.e. the turbine. This is followed by a mechanical connection 11 of reduction gear 10 to the crankshaft or the flywheel. The crankshaft or flywheel here would represent an input point for the flow of power. The only thing that is critical here is that the input point must be on the engine side of the clutch 2. Input into a transmission input shaft 3

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would make no sense, since with combustion engine **1** running it would then never be possible to bring about a non-driven condition.

The compound drive according to the invention can be described well by comparing FIG. **2** directly to FIG. **1**, which was just described. As may be seen from FIG. **2**, according to the invention the presence of a turbocharger **4** is not absolutely necessary. The only thing that is essential to the invention is that an exhaust line **5** be routed through compound drive turbine **7**. An interposed turbocharger **4** would be entirely optional. The presence of two reduction gears **8** and **10** is also not absolutely necessary. Only one reduction gear **8** or **10** is important, because it is necessary to match the high speed of compound drive turbine **7** to the speed of the crankshaft. This can be done either with reduction gear **8** between compound drive turbine **7** and torsion vibration damper **12**, or can also be realized between torsion vibration damper **12** and the mechanical connection **11** to the crankshaft or to the flywheel. But in another design of the invention it is also possible to employ both reduction gears **8** and **10**, as is known from the existing art.

Torsion vibration damper **12** shown schematically in FIG. **2** comprises a right-hand, disk-shaped input part and a left-hand output part, also disk-shaped. Located between the input and output parts is at least one extensively acting energy storage device, which may be in the form of a spring. In another design of the invention there is a plurality of extensively active energy storage devices present; these may be connected in parallel or in series. By designing torsion vibration damper **12** appropriately, it is possible here to influence the damping behavior and thus the frequency response curve. By designing energy storage devices having different spring characteristics, it is possible to generate overall spring characteristics that are either progressive or regressive in shape. In another design of the invention, the energy storage devices are situated at the outermost diameter of torsion vibration damper **12**. Here the energy storage devices (springs) may be guided in a sliding form. FIG. **6** depicts another design of the invention the springs **40** situated at the outermost diameter **12a** in which they are guided by means of sliding blocks **42**; that is, the springs **40** are provided on their radial outer side with sliding blocks **42**, which mesh between at least two turns of the springs **40** by means of a stop tab **42a**. In another design of this idea, also seen in FIG. **6**, there also rollers **44** situated between a slideway **46** located radially on the outside and the sliding blocks **42**, so that the friction is reduced.

The damping behavior of a torsion vibration damper **12** can be determined, that is adjusted or tuned, in a substantial way through dimensioning of frictions within the torsion vibration damper **12**. For that reason, in a first design of this invention the torsion vibration damper **12** can be provided with a lubrication of grease. In a second design the torsion vibration damper **12** is provided with a lubrication of oil; care must be taken to ensure that the oil is not thrown off. In another advantageous design of the invention, the oil lubrication can be designed as a component of the oil lubrication of combustion engine **1**. FIG. **7** shows engine **1** connected to oil channel **32** of shaft **16** by oil line **33** extending from engine **1** and return line **33a** extending from channel **32** to engine **1**.

FIG. **3** reveals a torsion vibration damper **12** that is equipped with extensively active energy storage devices. In this case these energy storage devices are springs **21**. Torque is introduced from compound drive turbine **7** through a gear wheel **14**. Gear wheel **14** is connected by a rotationally fixed connection to an additional mass **15**, which is situated on a shaft **16** by means of a bearing **17**. According to the invention, gear wheel **14** and additional mass **15** may also be executed in

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one piece. By means of a mounting designed for example as a riveted connection **18**, the flow of force into input part **19** takes place, where input part **19** comprises two disks that are riveted together. The flow of force then takes place from input part **19** to an output part **20** through the springs **21**. As can be seen from FIG. **3**, output part **20** touches left-hand input part **19**. This is achieved by positioning a diaphragm spring between output part **20** and the right-hand input part **19**. Depending on the dimensioning of this diaphragm spring, a defined friction then occurs between input part **19** and output part **20**, which can dissipate part of the vibrational energy. This metered friction can also be disadvantageous, however, because with the relative rotational motions between input part **19** and output part **20** there may thus be drag torque.

In another design of the invention, the attempt is made to keep the friction between input part **19** and output part **20** as low as possible. In that case one then also speaks of so-called vibration insulation. With vibration isolation the frequency response curve appears in a very narrow band, which has the advantage that the natural frequency of torsion vibration damper **12** can be defined more clearly or tunable, and can also be designed to fall clearly outside of the operating spectrum.

In another design of the invention there is an absorber **50** located between exhaust turbine **7** and the input point on the crankshaft or flywheel. This absorber can be designed so that it vibrates in the opposite phase.

FIG. **4** depicts a compound drive which is situated in a housing comprising housing parts **25**, **26**. Shaft **16** is guided by means of roller bearings **22**. The introduction of torque **13** takes place here at the larger gear wheel. A power take-off gear **23** has a smaller diameter. The explanation for this is that the speeds of rotation must be reduced from the high-speed compound drive turbine **7** to the mechanical connection **11** to the crankshaft.

As explained earlier, the flow of torque enters the compound drive through the torque input **13**. Torque input **13** is connected by means of a severable connection to a sleeve which is mounted on shaft **16** by means of two roller bearings **17**. Input part **19** of torsion vibration damper **12** is connected to the sleeve with a rotationally fixed connection. In the exemplary embodiment in FIG. **4**, the springs **21** of torsion vibration damper **12** are guided radially on the outside by means of a sliding form **24**. The output part **20** of torsion vibration damper **12** is connected in turn to shaft **16** by a rotationally fixed connection. Due to the precise positioning of both input part **19** and output part **20**—both axially and radially—it is possible for torsion vibration damper **12** to work precisely.

Shaft **16** has an oil channel **32** at its right end, which is depicted with dashed lines because of its hidden position. This oil channel **32** also has two transverse channels, through which oil can be directed both to torsion vibration damper **12** and to torque input **13**. The inlet for oil channel **32**—not shown here—can advantageously be situated in the area of right-hand bearing **22** in right-hand housing part **26**. It must also be mentioned that the angular ball bearings **17** are fixed in the sleeve or on shaft **16** by means of a lock nut **30** or retaining ring **29**.

FIG. **5** differs from FIG. **4** in that it is provided with a free wheeling mechanism **31**. This free wheeling mechanism **31** is designed so that it locks when there is a flow of torque from torque input **13** to take-off gear wheel **23**. In other words: When shaft **16** rotates faster than output part **20**, shaft **16** can turn freely. The free-wheeling is especially effective from the perspective of energy when the engine is being started or during acceleration. In these cases the combustion engine is

faster than exhaust turbine 7, because the exhaust turbine 7 needs some time before it reaches its optimal operating speed.

Since a torsion vibration damper 12 is usually operated with pulsation—that is, a basic load with undulation overlaid—in this case free wheeling mechanism 31 is not used. But if the basic load is small and the vibration amplitudes are correspondingly large, the vibrations can go beyond the zero position. In these cases the free wheeling mechanism 31 is also advantageous for the damping behavior of torsion vibration damper 12.

In the exemplary embodiment in FIG. 5, free wheeling mechanism 31 is situated between an outer sleeve and shaft 16. In this exemplary embodiment the rolling elements of free wheeling mechanism 31 do not run directly on shaft 16, however, but rather they run on an inner sleeve 27, which preferably has a hardened surface. This inner sleeve 27 is then fixed on the shaft 16 for example by means of a shrink joint.

REFERENCE LABELS

- 1 combustion engine (motor)
- 2 clutch
- 3 transmission input shaft
- 4 turbocharger
- 4a exhaust gas turbine
- 4b charge air turbine
- 5 exhaust line
- 5a exhaust line between engine and turbocharger
- 5b exhaust line between turbocharger and additional exhaust gas turbine (compound drive turbine)
- 6a charge air stream (between turbocharger and engine)
- 6b charge air stream (on the intake side of the turbocharger)
- 7 compound drive turbine
- 8 reduction bear (between compound drive turbine and Föttinger coupling)
- 9 Föttinger coupling
- 10 reduction gear
- 11 mechanical connection to the crankshaft or to the flywheel
- 12 torsion vibration damper/vibration insulator
- 13 torque input of compound drive turbine
- 14 gear wheel
- 15 additional mass
- 16 shaft
- 17 roller bearing
- 18 riveted connection
- 19 input part
- 20 output part
- 21 spring
- 22 roller bearing of shaft 16
- 23 power take-off gear
- 24 sliding form
- 25 housing part
- 26 housing part
- 27 inner sleeve
- 28 riveted connection
- 29 retaining ring
- 30 lock nut
- 31 free wheeling mechanism
- 32 oil channel

What is claimed is:

1. A compound drive comprising in combination a combustion engine and at least one exhaust gas turbine in the exhaust gas stream and an absorber, the absorber located between said exhaust turbine and an input point, where the energy of the exhaust gas turbine in the flow of power from the combustion engine is fed into said input point, there being a

speed reduction gear situated between the exhaust gas turbine and the input point, characterized in that a torsion vibration damper—having at least one input part and output part—is situated in the flow of force from the exhaust gas turbine to the input point;

wherein said torsion vibration damper is equipped with extensively active energy storage devices;

wherein said extensively active energy storage devices are in the form of coil springs;

wherein said input part of said torsion vibration damper is radially outwardly positioned over the input point relative to the output part of said torsion vibration damper and,

wherein the absorber tunes the damping behavior of the torsion vibration damper.

2. The compound drive according to claim 1, wherein a turbocharger is situated in the exhaust gas stream between the exhaust gas turbine and the combustion engine.

3. The compound drive according to claim 1 wherein an additional mass is connected to the at least one input part with a rotationally fixed connection.

4. The compound drive according to claim 1, wherein an additional mass is connected to the at least one output part with a rotationally fixed connection.

5. The compound drive according to claim 1, wherein the springs of the torsion vibration damper—at least in part—are extensively guided at their outside diameter in a sliding form.

6. The compound drive according to claim 1, wherein said springs of the torsion vibration damper—at least in part—are extensively guided at their outside diameter by sliding blocks, said sliding blocks being fixed between two turns of said springs by means of a stop tab.

7. The compound drive according to claim 6, wherein said sliding blocks are provided with rollers, said rollers being situated between said sliding blocks and an ample slideway.

8. The compound drive according to claim 1, wherein said energy storage devices having different spring characteristics are combined.

9. The compound drive according to claim 1, wherein said torsion vibration damper is equipped with an oil lubrication system.

10. The compound drive according to claim 9, wherein said oil lubrication system is coupled with an oil lubrication system of said combustion engine.

11. The compound drive according to claim 1, wherein a natural frequency of said torsion vibration damper is tunable.

12. The compound drive according to claim 1, wherein a flow of force coming from said torsion vibration damper is fed into a flywheel.

13. The compound drive according to claim 1, wherein the flow of force coming from the torsion vibration damper is fed into a crankshaft.

14. The compound drive according to claim 1, wherein said torsion vibration damper is equipped with a free wheeling mechanism.

15. The compound drive according to claim 1, wherein said torsion vibration damper is a vibration insulator.

16. The compound drive according to claim 1, wherein an absorber is situated in the power train between said exhaust gas turbine and said input point.

17. The compound drive according to claim 1, wherein said torsion vibration damper comprises at least one rotationally acting energy storage device acting on rolling elements that roll on configured slideways.